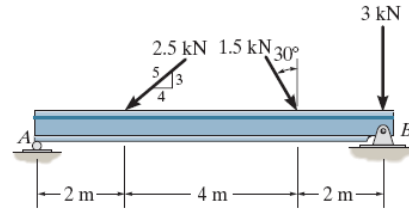


## Homework Solutions # 6

4-106. Replace the force system acting on the beam by an equivalent force and couple moment at point B.



$$\begin{aligned} \rightarrow F_{R_x} = \Sigma F_x; \quad F_{R_x} &= 1.5 \sin 30^\circ - 2.5 \left(\frac{4}{5}\right) \\ &= -1.25 \text{ kN} = 1.25 \text{ kN} \leftarrow \end{aligned}$$

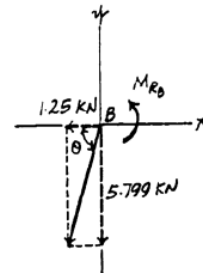
$$\begin{aligned} + \uparrow F_{R_y} = \Sigma F_y; \quad F_{R_y} &= -1.5 \cos 30^\circ - 2.5 \left(\frac{3}{5}\right) - 3 \\ &= -5.799 \text{ kN} = 5.799 \text{ kN} \downarrow \end{aligned}$$

Thus,

$$F_R = \sqrt{F_{R_x}^2 + F_{R_y}^2} = \sqrt{1.25^2 + 5.799^2} = 5.93 \text{ kN} \quad \text{Ans}$$

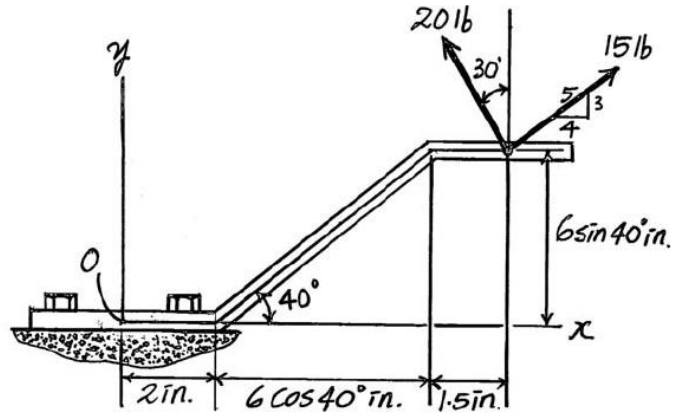
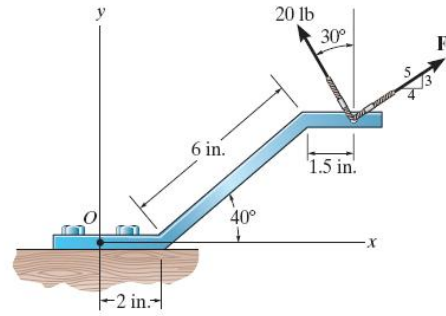
and

$$\theta = \tan^{-1} \left( \frac{F_{R_y}}{F_{R_x}} \right) = \tan^{-1} \left( \frac{5.799}{1.25} \right) = 77.8^\circ \quad \text{Ans}$$



$$\begin{aligned} \curvearrowright + M_{R_B} = \Sigma M_B; \quad M_{R_B} &= 1.5 \cos 30^\circ (2) + 2.5 \left(\frac{3}{5}\right) (6) \\ &= 11.6 \text{ kN} \cdot \text{m} \quad (\text{Counterclockwise}) \quad \text{Ans} \end{aligned}$$

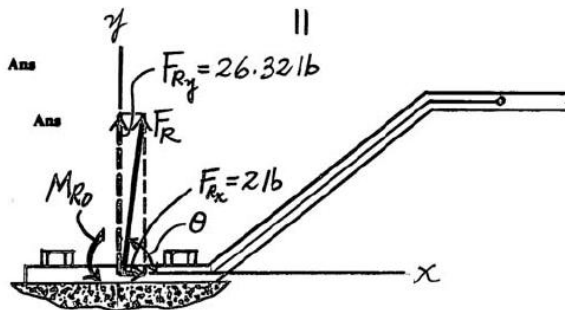
\*4-108. Replace the two forces by an equivalent resultant force and couple moment at point  $O$ . Set  $F = 15$  lb.



$$\begin{aligned} \rightarrow F_{Rx} = \Sigma F_x; \quad F_{Rx} &= \frac{4}{5}(15) - 20 \sin 30^\circ = 2 \text{ lb} \\ + \uparrow F_{Ry} = \Sigma F_y; \quad F_{Ry} &= 20 \cos 30^\circ + \frac{3}{5}(15) = 26.32 \text{ lb} \end{aligned}$$

$$\begin{aligned} F_R &= \sqrt{F_{Rx}^2 + F_{Ry}^2} = \sqrt{2^2 + 26.32^2} = 26.4 \text{ lb} \\ \theta &= \tan^{-1} \frac{F_{Ry}}{F_{Rx}} = \tan^{-1} \left( \frac{26.32}{2} \right) = 85.7^\circ \end{aligned}$$

$$\begin{aligned} (+ M_{Ro} = \Sigma M_O; \quad M_{Ro} &= 20 \sin 30^\circ (6 \sin 40^\circ) + 20 \cos 30^\circ (3.5 + 6 \cos 40^\circ) \\ &\quad - \frac{4}{5}(15)(6 \sin 40^\circ) + \frac{3}{5}(15)(3.5 + 6 \cos 40^\circ) \\ &= 205 \text{ lb} \cdot \text{in.} \end{aligned}$$



4-110. Replace the force and couple moment system acting on the overhang beam by a resultant force and couple moment at point A.

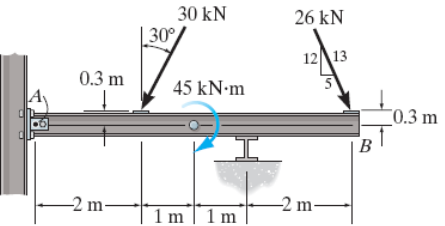
**Equivalent Resultant Force:** Forces  $F_1$  and  $F_2$  are resolved into their  $x$  and  $y$  components, Fig. *a*. Summing these force components algebraically along the  $x$  and  $y$  axes,

$$\rightarrow \Sigma (F_R)_x = \Sigma F_x: (F_R)_x = 26\left(\frac{5}{13}\right) - 30 \sin 30^\circ = -5 \text{ kN} = 5 \text{ kN} \leftarrow$$

$$+ \uparrow (F_R)_y = \Sigma F_y: (F_R)_y = -26\left(\frac{12}{13}\right) - 30 \cos 30^\circ = -49.98 \text{ kN} = 49.98 \text{ kN} \downarrow$$

The magnitude of the resultant force  $F_R$  is given by

$$F_R = \sqrt{(F_R)_x^2 + (F_R)_y^2} = \sqrt{5^2 + 49.98^2} = 50.23 \text{ kN} = 50.2 \text{ kN}$$



Ans.

The angle  $\theta$  of  $F_R$  is

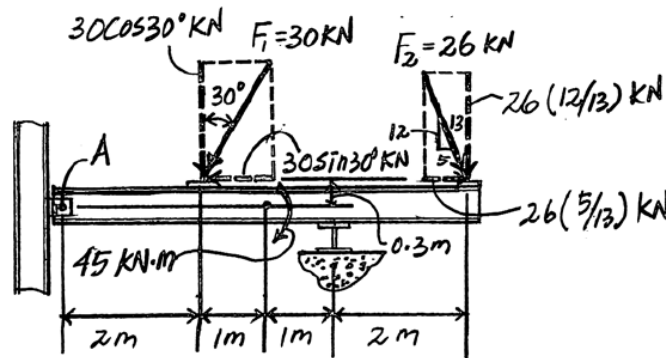
$$\theta = \tan^{-1} \left[ \frac{(F_R)_y}{(F_R)_x} \right] = \tan^{-1} \left[ \frac{49.98}{5} \right] = 84.29^\circ = 84.3^\circ \swarrow$$

Ans.

**Equivalent Resultant Couple Moment:** Applying the principle of moments, Figs. *a* and *b*, and summing the moments of the force components algebraically about point A,

$$\begin{aligned} \curvearrowleft (M_R)_A = \Sigma M_A: (M_R)_A &= 30 \sin 30^\circ (0.3) - 30 \cos 30^\circ (2) - 26\left(\frac{5}{13}\right)(0.3) - 26\left(\frac{12}{13}\right)(6) - 45 \\ &= -239.46 \text{ kN} \cdot \text{m} = 239 \text{ kN} \cdot \text{m} \text{ (clockwise)} \end{aligned}$$

Ans.



(a)

4-130. The building slab is subjected to four parallel column loadings. Determine the equivalent resultant force and specify its location  $(x, y)$  on the slab. Take  $F_1 = 20$  kN,  $F_2 = 50$  kN.

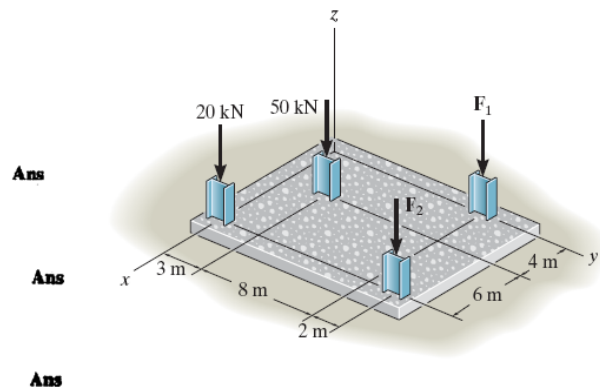
$$+\downarrow F_R = \Sigma F_z; \quad F_R = 20 + 50 + 20 + 50 = 140 \text{ kN}$$

$$M_{R_y} = \Sigma M_y; \quad 140(x) = (50)(4) + 20(10) + 50(10)$$

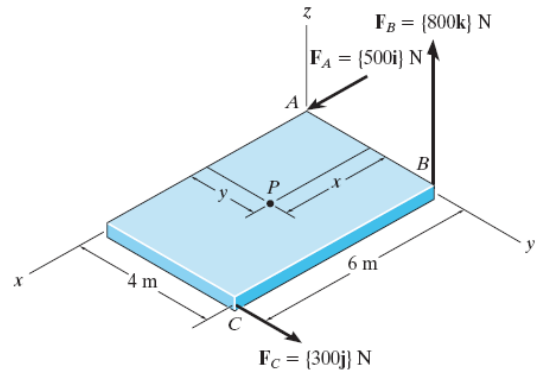
$$x = 6.43 \text{ m}$$

$$M_{R_x} = \Sigma M_x; \quad -140(y) = -(50)(3) - 20(11) - 50(13)$$

$$y = 7.29 \text{ m}$$



•4-141. Replace the three forces acting on the plate by a wrench. Specify the magnitude of the force and couple moment for the wrench and the point  $P(x, y)$  where its line of action intersects the plate.



$$\mathbf{F}_R = \{500\mathbf{i} + 300\mathbf{j} + 800\mathbf{k}\} \text{ N}$$

$$F_R = \sqrt{(500)^2 + (300)^2 + (800)^2} = 990 \text{ N} \quad \text{Ans}$$

$$\mathbf{u}_{FR} = \{0.5051\mathbf{i} + 0.3030\mathbf{j} + 0.8081\mathbf{k}\}$$

$$M_{R_x'} = \Sigma M_{x'}; \quad M_{R_x'} = 800(4-y)$$

$$M_{R_y'} = \Sigma M_{y'}; \quad M_{R_y'} = 800x$$

$$M_{R_z'} = \Sigma M_{z'}; \quad M_{R_z'} = 500y + 300(6-x)$$

Since  $M_R$  also acts in the direction of  $\mathbf{u}_{FR}$ ,

$$M_R(0.5051) = 800(4-y)$$

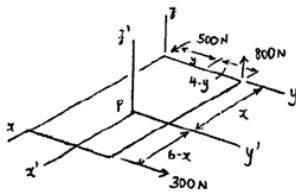
$$M_R(0.3030) = 800x$$

$$M_R(0.8081) = 500y + 300(6-x)$$

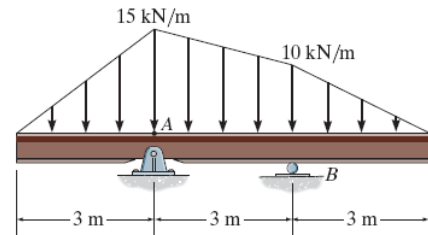
$$M_R = 3.07 \text{ kN}\cdot\text{m} \quad \text{Ans}$$

$$x = 1.16 \text{ m} \quad \text{Ans}$$

$$y = 2.06 \text{ m} \quad \text{Ans}$$



4-142. Replace the distributed loading with an equivalent resultant force, and specify its location on the beam measured from point A.



**Loading:** The distributed loading can be divided into four parts as shown in Fig. *a*. The magnitude and location of the resultant force of each part acting on the beam are also indicated in Fig. *a*.

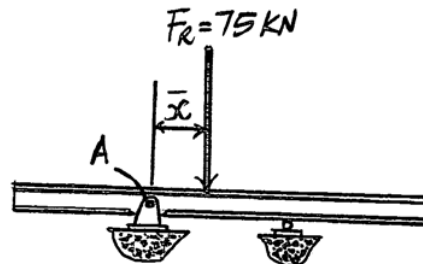
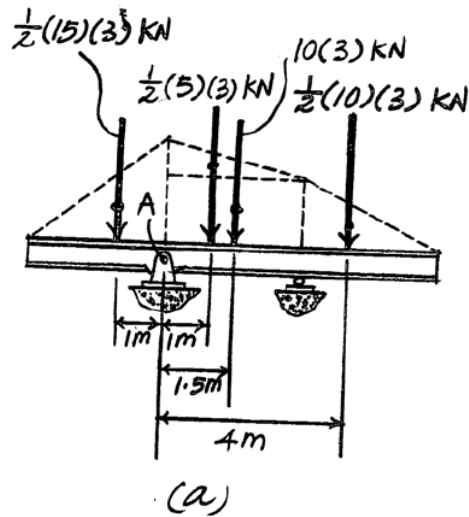
**Resultants:** Equating the sum of the forces along the *y* axis of Figs. *a* and *b*,

$$+\downarrow F_R = \Sigma F_y; \quad F_R = \frac{1}{2}(15)(3) + \frac{1}{2}(5)(3) + 10(3) + \frac{1}{2}(10)(3) = 75 \text{ kN} \downarrow \quad \text{Ans.}$$

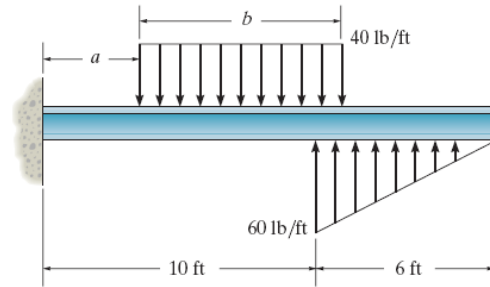
If we equate the moments of  $F_R$ , Fig. *b*, to the sum of the moment of the forces in Fig. *a* about point A,

$$\curvearrowleft + (M_R)_A = \Sigma M_A; \quad -75(\bar{x}) = \frac{1}{2}(15)(3)(1) - \frac{1}{2}(5)(3)(1) - 10(3)(1.5) - \frac{1}{2}(10)(3)(4)$$

$$\bar{x} = 1.20 \text{ m} \quad \text{Ans.}$$



4-150. The beam is subjected to the distributed loading. Determine the length  $b$  of the uniform load and its position  $a$  on the beam such that the resultant force and couple moment acting on the beam are zero.



Require  $F_R = 0$ .

$$+\uparrow F_R = \Sigma F_y; \quad 0 = 180 - 40b$$

$$b = 4.50 \text{ ft}$$

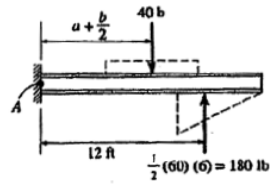
Ans

Require  $M_{R_A} = 0$ . Using the result  $b = 4.50$  ft, we have

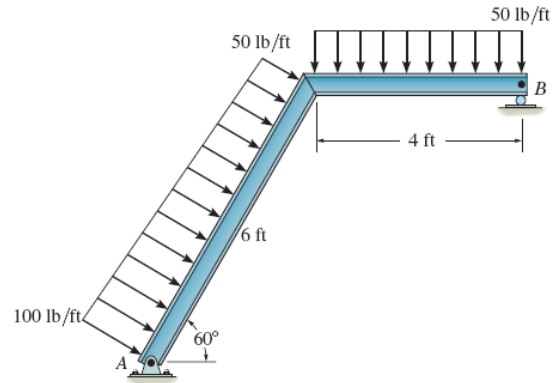
$$\curvearrowleft +M_{R_A} = \Sigma M_A; \quad 0 = 180(12) - 40(4.50) \left( a + \frac{4.50}{2} \right)$$

$$a = 9.75 \text{ ft}$$

Ans



\*4-156. Replace the loading by an equivalent resultant force and couple moment acting at point B.



$$F_1 = \frac{1}{2} (6) (50) = 150 \text{ lb}$$

$$F_2 = (6) (50) = 300 \text{ lb}$$

$$F_3 = (4) (50) = 200 \text{ lb}$$

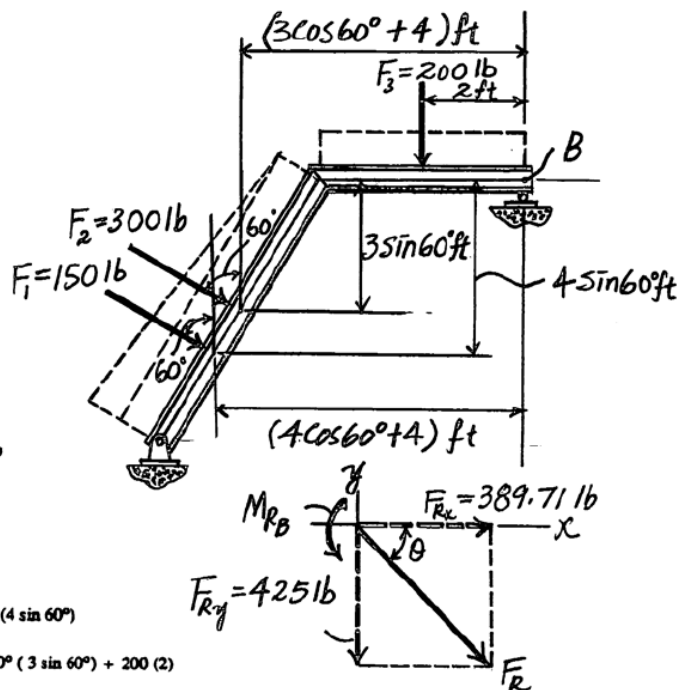
$$\rightarrow F_{Rx} = \Sigma F_x; \quad F_{Rx} = 150 \sin 60^\circ + 300 \sin 60^\circ = 389.71 \text{ lb}$$

$$+\downarrow F_{Ry} = \Sigma F_y; \quad F_{Ry} = 150 \cos 60^\circ + 300 \cos 60^\circ + 200 = 425 \text{ lb}$$

$$F_R = \sqrt{(389.71)^2 + (425)^2} = 577 \text{ lb} \quad \text{Ans}$$

$$\theta = \tan^{-1} \left( \frac{425}{389.71} \right) = 47.5^\circ \quad \text{Ans}$$

$$\begin{aligned} (+M_{RB} = \Sigma M_B; \quad M_{RB} &= 150 \cos 60^\circ (4 \cos 60^\circ + 4) + 150 \sin 60^\circ (4 \sin 60^\circ) \\ &\quad + 300 \cos 60^\circ (3 \cos 60^\circ + 4) + 300 \sin 60^\circ (3 \sin 60^\circ) + 200 (2) \\ M_{RB} &= 2800 \text{ lb} \cdot \text{ft} = 2.80 \text{ kip} \cdot \text{ft} \quad \text{Ans} \end{aligned}$$





\*4-168. Determine the magnitude of the moment of the force  $F_C$  about the hinged axis  $aa$  of the door.

$$\mathbf{r}_{AB} = \{[-0.5 - (-0.5)]\mathbf{i} + [0 - (-1)]\mathbf{j} + (0 - 0)\mathbf{k}\} \text{ m} = \{1\mathbf{j}\} \text{ m}$$

$$\mathbf{F}_C = 250 \left( \frac{[-0.5 - (-2.5)]\mathbf{i} + [0 - (-(1 + 1.5\cos 30^\circ))]\mathbf{j} + (0 - 1.5\sin 30^\circ)\mathbf{k}}{\sqrt{[-0.5 - (-2.5)]^2 + [0 - (-(1 + 1.5\cos 30^\circ))]^2 + (0 - 1.5\sin 30^\circ)^2}} \right) \text{ N}$$

$$= \{159.33\mathbf{i} + 183.15\mathbf{j} - 59.75\mathbf{k}\} \text{ N}$$

**Moment of Force  $F_C$  About  $a$ - $a$  Axis:** The unit vector along the  $a$ - $a$  axis is  $\mathbf{i}$ .  
Applying Eq. 4-11, we have

$$M_{a-a} = \mathbf{i} \cdot (\mathbf{r}_{AB} \times \mathbf{F}_C)$$

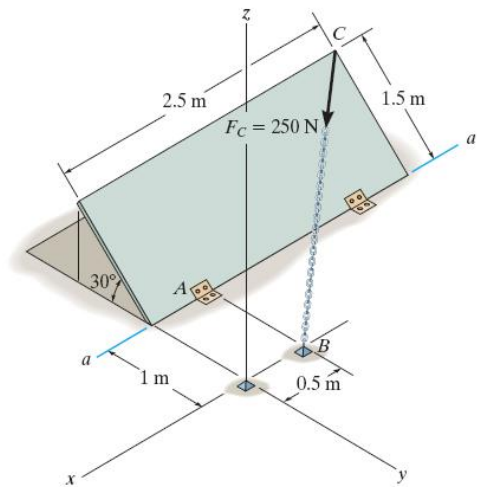
$$= \begin{vmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 159.33 & 183.15 & -59.75 \end{vmatrix}$$

$$= 1[1(-59.75) - (183.15)(0)] - 0 + 0$$

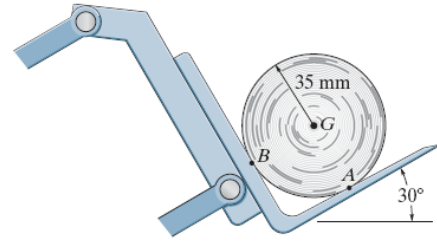
$$= -59.7 \text{ N} \cdot \text{m}$$

The negative sign indicates that  $M_{a-a}$  is directed toward negative  $x$  axis.  
 $M_{a-a} = 59.7 \text{ N} \cdot \text{m}$

**Ans**



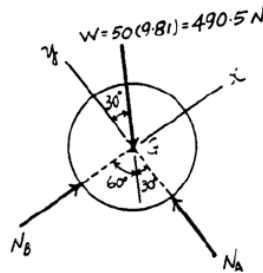
•5-1. Draw the free-body diagram of the 50-kg paper roll which has a center of mass at  $G$  and rests on the smooth blade of the paper hauler. Explain the significance of each force acting on the diagram. (See Fig. 5-7b.)



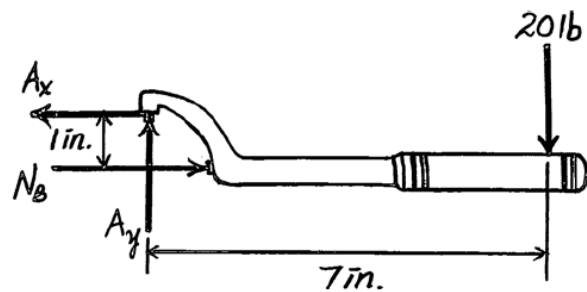
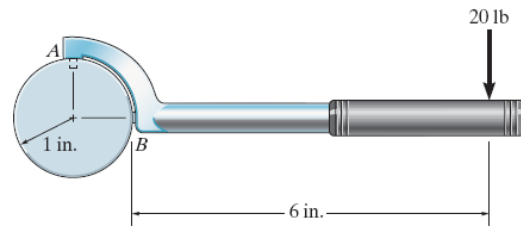
**The Significance of Each Force :**

$W$  is the effect of gravity (weight) on the paper roll.

$N_A$  and  $N_B$  are the smooth blade reactions on the paper roll.



5-7. Draw the free-body diagram of the “spanner wrench” subjected to the 20-lb force. The support at  $A$  can be considered a pin, and the surface of contact at  $B$  is smooth. Explain the significance of each force on the diagram. (See Fig. 5-7b.)



$A_x, A_y, N_B$  force of cylinder on wrench.